



Design against Fluctuating Load Formulas

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9) Fatigue Stress Concentration Factor given Modifying Factor 🕑



12) Reliability Factor for Fluctuating Load 🖸

 $\mathbf{f_X} \mathbf{K_c} = \frac{\mathbf{S_e}}{\mathbf{S'_e} \cdot \mathbf{K_d} \cdot \mathbf{K_a} \cdot \mathbf{K_b}}$ $\mathbf{e_X} \mathbf{0.87189} = \frac{51 \mathrm{N/mm^2}}{220 \mathrm{N/mm^2} \cdot 0.34 \cdot 0.92 \cdot 0.85}$

Open Calculator 🕑





$$\label{eq:Kb} \begin{array}{l} \mbox{ } \textbf{K}_b = \frac{S_e}{S'_e \cdot K_d \cdot K_c \cdot K_a} \end{array} \end{array} \begin{array}{l} \mbox{ } \textbf{Open Calculator C} \end{array}$$

14) Stress Amplitude for Fluctuating load given Maximum Stress and Minimum Stress







Notch Sensitivity for Fluctuating Loads C

16) Fatigue Stress Concentration Factor

$$\mathbf{fx} \ \mathbf{k}_{f} = \frac{Se_{nf}}{Se_{n}}$$
Open Calculator (*)
$$\mathbf{k}_{f} = \frac{Se_{nf}}{Se_{n}}$$

$$\mathbf{ex} \ 2.277778 = \frac{205N/mm^{2}}{90N/mm^{2}}$$

$$\mathbf{17} \ \mathbf{Fatigue \ Stress \ Concentration \ Factor \ given \ Notch \ Sensitivity \ Factor \ (*)
$$\mathbf{k}_{f} = 1 + \mathbf{q} \cdot (\mathbf{k}_{t} - 1)$$
Open Calculator (*)
$$\mathbf{ex} \ 2 = 1 + 0.5 \cdot (3 - 1)$$

$$\mathbf{18} \ \mathbf{Notch \ Sensitivity \ Factor \ (*)}$$

$$\mathbf{ex} \ \mathbf{q} = \frac{\sigma_{an}}{\sigma_{tn}}$$

$$\mathbf{ex} \ 0.727273 = \frac{80N/mm^{2}}{110N/mm^{2}}$$

$$\mathbf{19} \ \mathbf{Notch \ Sensitivity \ Factor \ given \ Fatigue \ Stress \ Concentration \ Factor \ (*)$$$$



Open Calculator 🕑

$$\begin{array}{c} \mbox{7/24}\\ \mbox{20) Theoretical Stress for Fluctuating Load } \label{eq:stress} \end{array} \\ \mbox{20) Theoretical Stress for Fluctuating Load } \label{eq:stress} \end{array} \\ \mbox{20) Theoretical Stress for Fluctuating Load } \label{eq:stress} \\ \mbox{21) For } \mbox{21) Goodman Line Amplitude Stress } \mbox{22) Goodman Line Endurance Limit } \mbox{23) } \mbox{24) } \mbox{24) } \mbox{26) } \mbox{26) } \mbox{26) } \mbox{27) } \mbox{2$$

ex
$$33.84615 \mathrm{N/mm^2} = rac{30 \mathrm{N/mm^2}}{1 - rac{50 \mathrm{N/mm^2}}{440 \mathrm{N/mm^2}}}$$







27) Permissible Mean Stress for Fluctuating Load 🕑



28) Permissible Stress Amplitude for Fluctuating Load

fx
$$\sigma_{\rm a}={S_{\rm a}\over f_{\rm s}}$$
 Open Calculator C (C) $\sigma_{\rm a}={S_{\rm a}\over f_{\rm s}}$

29) Slope of Line OE in Modified Goodman Diagram given Bending Amplitude and Mean Bending Moment

fx
$$m = rac{M_{ba}}{M_{bm}}$$

ex $1.5 = rac{1800 \mathrm{N}^*\mathrm{mm}}{1200 \mathrm{N}^*\mathrm{mm}}$

Open Calculator 🕑



Open Calculator

Open Calculator

30) Slope of Line OE in Modified Goodman Diagram given Force Amplitude and Mean Force

fx
$$m = \frac{P_a}{P_m}$$
 Open Calculator IP
ex $0.342105 = \frac{26N}{76N}$

31) Slope of Line OE in Modified Goodman Diagram given Stress Amplitude and Mean Stress

fx
$$m = rac{\sigma_a}{\sigma_m}$$
 ex $0.6 = rac{30
m N/mm^2}{50
m N/mm^2}$

32) Soderberg Line Amplitude Stress 🖸

fx
$$\sigma_{\rm a} = {
m S}_{
m e} \cdot \left(1 - rac{\sigma_{
m m}}{\sigma_{
m yt}}
ight)$$

ex $21 {
m N/mm^2} = 51 {
m N/mm^2} \cdot \left(1 - rac{50 {
m N/mm^2}}{85 {
m N/mm^2}}
ight)$





33) Soderberg Line Endurance Limit 子







Stress Concentration Factors in Design C

36) Diameter of Shaft given Ratio of Torsional Strength of Shaft with Keyway to without Keyway

fx
$$\mathbf{d} = rac{0.2 \cdot \mathbf{b}_{\mathrm{k}} + 1.1 \cdot \mathbf{h}}{1 - \mathrm{C}}$$

$$45\text{mm} = \frac{0.2 \cdot 5\text{mm} + 1.1 \cdot 4\text{mm}}{1 - 0.88}$$

37) Height of Shaft Keyway given Ratio of Torsional Strength of Shaft with Keyway to without Keyway

$$f_{X} h = \frac{d}{1.1} \cdot \left(1 - C - 0.2 \cdot \frac{b_{k}}{d}\right)$$

$$e_{X} 4mm = \frac{45mm}{1.1} \cdot \left(1 - 0.88 - 0.2 \cdot \frac{5mm}{45mm}\right)$$

$$(1 - 0.88 - 0.2 \cdot \frac{5mm}{45mm})$$

$$(38) Highest Value of Actual Stress near Discontinuity$$

$$f_{X} \sigma a_{max} = k_{f} \cdot \sigma_{o}$$

$$Open Calculator$$

$$x 53.75 N/mm^2 = 2.15 \cdot 25 N/mm^2$$



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Open Calculator







 $32 \mathrm{mm}$

3 - 1

ex 16mm =

43) Width of Shaft Keyway given Ratio of Torsional Strength of Shaft with Keyway to without Keyway

$$f_{X} b_{k} = 5 \cdot d \cdot \left(1 - C - 1.1 \cdot \frac{h}{d}\right)$$

$$f_{X} b_{k} = 5 \cdot d \cdot \left(1 - C - 1.1 \cdot \frac{h}{d}\right)$$

$$f_{X} 5mm = 5 \cdot 45mm \cdot \left(1 - 0.88 - 1.1 \cdot \frac{4mm}{45mm}\right)$$
Flat Plate against Fluctuating Loads **C**

$$f_{X} P = \sigma_{o} \cdot d_{o} \cdot t$$

$$f_{X} P = \sigma_{o} \cdot d_{o} \cdot d_{o}$$



47) Nominal Tensile Stress in Flat Plate with Shoulder Fillet 🕑



ex	$10 \mathrm{mm} =$	8750N
		$25 \mathrm{N/mm^2} \cdot 35 \mathrm{mm}$





Rectangular Plate against Fluctuating Loads 🖸

50) Diameter of Transverse Hole of Rectangular Plate with Stress Concentration given Nominal Stress





16/24

53) Thickness of Rectangular Plate with Transverse Hole given Nominal Stress







56) Nominal Bending Stress in Round Shaft with Shoulder Fillet 🕑



57) Nominal Tensile Stress in Round Shaft with Shoulder Fillet 🕑



58) Nominal Torsional Stress in Round Shaft with Shoulder Fillet

$$\begin{aligned} & \mathbf{fx} \mathbf{\sigma}_{\mathrm{o}} = \frac{16 \cdot (\mathrm{M_{t}t})}{\pi \cdot \mathrm{d}_{\mathrm{small}}^{3}} \end{aligned}$$

$$\mathbf{ex} 24.88843 \mathrm{N/mm^{2}} = \frac{16 \cdot 28500 \mathrm{N*mm}}{\pi \cdot (18 \mathrm{mm})^{3}} \end{aligned}$$





Open Calculator

59) Smaller Diameter of Round Shaft with Shoulder Fillet in Tension or Compression

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19/24

ex
$$21.11004 \mathrm{mm} = \sqrt{rac{4 \cdot 8750 \mathrm{N}}{\pi \cdot 25 \mathrm{N/mm^2}}}$$

 $\left| \frac{4 \cdot P}{\pi \cdot \sigma_0} \right|$

 $\mathrm{d}_{\mathrm{small}} =$

60) Tensile Force in Round Shaft with Shoulder Fillet given Nominal Stress

fx
$$P = \frac{\sigma_o \cdot \pi \cdot d_{small}^2}{4}$$

ex $6361.725N = \frac{25N/mm^2 \cdot \pi \cdot (18mm)^2}{4}$

61) Torsional Moment in Round Shaft with Shoulder Fillet given Nominal Stress 🖸

$$f_{\mathbf{X}}\left(\mathbf{M}_{t}\mathbf{t}\right) = \frac{\tau_{o} \cdot \pi \cdot \mathbf{d}_{small}^{3}}{16}$$
ex 22902.21N*mm =
$$\frac{20N/mm^{2} \cdot \pi \cdot (18mm)^{3}}{16}$$



Variables Used

- **a**_e Major Axis of Elliptical Crack (*Millimeter*)
- **b**_e Minor Axis of Elliptical Crack (Millimeter)
- **b**_k Width of Key in Round Shaft (*Millimeter*)
- C Ratio of Shaft Strength with and without Keyway
- **d** Diameter of Shaft with Keyway (Millimeter)
- **d**_h Diameter of Transverse Hole in Plate (*Millimeter*)
- **d**_o Smaller Width of Plate (*Millimeter*)
- dsmall Smaller Diameter of Shaft with Fillet (Millimeter)
- **f**_s Design Factor of Safety
- h Height of Shaft Keyway (Millimeter)
- Ka Surface Finish Factor
- K_b Size Factor
- K_c Reliabilty Factor
- Kd Modifying Factor for Stress Concentration
- k_f Fatigue Stress Concentration Factor
- kt Theoretical Stress Concentration Factor
- **m** Slope of modified Goodman Line
- Mb Bending Moment on Round Shaft (Newton Millimeter)
- Mba Bending Moment Amplitude (Newton Millimeter)
- Mbm Mean Bending Moment (Newton Millimeter)
- M_tt Torsional moment on Round Shaft (Newton Millimeter)

- P Load on Flat Plate (Newton)
- Pa Force Amplitude for Fluctuating Stress (Newton)
- Pm Mean Force for Fluctuating Stress (Newton)
- **q** Notch Sensitivity Factor
- **S**_a Limiting Value of Stress Amplitude (Newton per Square Millimeter)
- **S**_e Endurance Limit (Newton per Square Millimeter)
- **S'**_e Endurance Limit of Rotating Beam Specimen (*Newton per Square Millimeter*)
- **S**_{ea} Endurance Limit for Axial Loading (Newton per Square Millimeter)
- **S_m** Limiting Value of Mean Stress (*Newton per Square Millimeter*)
- **Se**_n Endurance Limit of Notched Specimen (Newton per Square *Millimeter*)
- **Se_{nf}** Endurance Limit of Notch Free Specimen (Newton per Square *Millimeter*)
- t Thickness of Plate (Millimeter)
- **W** Width of Plate (Millimeter)
- **σ** Theoretical Stress (Newton per Square Millimeter)
- σ_a Stress Amplitude for Fluctuating Load (Newton per Square Millimeter)
- σ_{an} Increase of Actual Stress over Nominal Stress (Newton per Square Millimeter)
- σ_m Mean Stress for Fluctuating Load (Newton per Square Millimeter)
- σ_{max fl} Maximum Stress Value for Fluctuating Load (Newton per Square Millimeter)
- σ_{max} Maximum Stress at Crack Tip (Newton per Square Millimeter)





- σ_{min fl} Minimum Stress Value for Fluctuating Load (Newton per Square Millimeter)
- σ_{min} Minimum Stress at Crack Tip (Newton per Square Millimeter)
- **σ**_o Nominal Stress (Newton per Square Millimeter)
- σ_{tn} Increase of Theoretical Stress over Nominal Stress (Newton per Square Millimeter)
- σ_{ut} Ultimate Tensile strength (Newton per Square Millimeter)
- σ_{yt} Tensile Yield Strength for Fluctuating load (Newton per Square Millimeter)
- σa_{max} Highest Value of Actual Stress near Discontinuity (Newton per Square Millimeter)
- T_o Nominal Torsional Stress for Fluctuating Load (Newton per Square Millimeter)



Constants, Functions, Measurements used

- Constant: pi, 3.14159265358979323846264338327950288 Archimedes' constant
- Function: **sqrt**, sqrt(Number) Square root function
- Measurement: Length in Millimeter (mm) Length Unit Conversion
- Measurement: Force in Newton (N) Force Unit Conversion
- Measurement: Torque in Newton Millimeter (N*mm)
 Torque Unit Conversion
- Measurement: Stress in Newton per Square Millimeter (N/mm²) Stress Unit Conversion

Check other formula lists

- Design against Fluctuating Load Design of Helical Gears Formulas 🗖 Formulas 🗖 Design of Bevel Gear Design of Keys Formulas Formulas 🕅 • Design of Knuckle Joint Formulas C Design of Chain Drives Formulas 🚰 Design of Lever Formulas Design of Cotter Joint Design of Pressure Vessels Formulas Formulas Design of Coupling Formulas C Design of Shafts Formulas Design of Flywheel Formulas Design of Threaded Fasteners Formulas C **Design of Friction Clutches** Formulas 🚰 Power Screws Formulas
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