



# **Friction Devices Formulas**

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### List of 26 Friction Devices Formulas

#### Pivot Bearing 🕑



fx 
$$T=rac{\mu_{
m f}\cdot W_{
m t}\cdot D_{
m s}\cdot h_{
m s}}{3}$$

ex 
$$2.4N^*m = \frac{0.4 \cdot 24N \cdot 0.5m \cdot 1.5m}{3}$$

#### 2) Frictional Torque on Conical Pivot Bearing by Uniform Wear 子

fx 
$$T = rac{\mu_{\mathrm{f}} \cdot \mathrm{W}_{\mathrm{t}} \cdot \mathrm{D}_{\mathrm{s}} \cdot \cos e c rac{lpha}{2}}{2}$$

ex 2.379418N\*m = 
$$\frac{0.4 \cdot 24N \cdot 0.5m \cdot \cos ec \frac{30.286549^{\circ}}{2}}{2}$$

#### 3) Frictional Torque on Flat Pivot Bearing by Uniform Pressure

fx 
$$\mathrm{T}=rac{2}{3}\cdot \mu_{\mathrm{f}}\cdot \mathrm{W}_{\mathrm{t}}\cdot \mathrm{R}$$

ex 
$$21.12N*m = \frac{2}{3} \cdot 0.4 \cdot 24N \cdot 3.3m$$



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# 4) Frictional Torque on Truncated Conical Pivot Bearing by Uniform Pressure

fx 
$$\mathbf{T}=rac{2}{3}\cdot \mu_{\mathrm{f}}\cdot \mathbf{W}_{\mathrm{t}}\cdot rac{\mathbf{r}_{1}^{3}-\mathbf{r}_{2}^{3}}{\mathbf{r}_{1}^{2}-\mathbf{r}_{2}^{2}}$$

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ex 
$$67.65714$$
N\*m =  $\frac{2}{3} \cdot 0.4 \cdot 24$ N  $\cdot \frac{(8m)^3 - (6m)^3}{(8m)^2 - (6m)^2}$ 

fx 
$$R_c = rac{R_1 + R_2}{2}$$
 ex  $0.04 \mathrm{m} = rac{0.050 \mathrm{m} + 0.03 \mathrm{m}}{2}$ 

#### 6) Pressure over Bearing Area of Flat Pivot Bearing 🗹

$$f_{X} \ p_{i} = \frac{W_{t}}{\pi \cdot R^{2}}$$

$$e_{X} \ 0.701509Pa = \frac{24N}{\pi \cdot (3.3m)^{2}}$$
7) Torque Required to Overcome Friction at Collar C
$$f_{X} \ T = \mu_{c} \cdot W_{1} \cdot R_{c}$$
Open Calculator C

ex 
$$0.1696N^*m = 0.16 \cdot 53N \cdot 0.02m$$





### 8) Total Frictional Torque on Conical Pivot Bearing Considering Uniform Pressure

fx 
$$\mathbf{T} = \mu_{\mathrm{f}} \cdot \mathbf{W}_{\mathrm{t}} \cdot \mathbf{D}_{\mathrm{s}} \cdot \cos e c rac{lpha}{3}$$

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ex 
$$3.172558$$
N\*m =  $0.4 \cdot 24$ N  $\cdot 0.5$ m  $\cdot \cos ec \frac{30.286549^{\circ}}{3}$ 

# 9) Total Frictional Torque on Conical Pivot Bearing Considering Uniform Wear when Slant Height of Cone

fx 
$$T = \frac{\mu_f \cdot W_t \cdot h_s}{2}$$
  
ex  $7.2N^*m = \frac{0.4 \cdot 24N \cdot 1.5m}{2}$ 

# 10) Total Frictional Torque on Flat Pivot Bearing Considering Uniform Wear

fx 
$$T = \frac{\mu_f \cdot W_t \cdot R}{2}$$
 Open Calculator (\*)  
ex  $15.84N^*m = \frac{0.4 \cdot 24N \cdot 3.3m}{2}$ 



# 11) Total Frictional Torque on Truncated Conical Pivot Bearing Considering Uniform Wear

fx 
$$\mathbf{T} = \mu_{\mathrm{f}} \cdot \mathbf{W}_{\mathrm{t}} \cdot rac{\mathbf{r}_1 + \mathbf{r}_2}{2}$$

$$\texttt{ex} \ 67.2 \texttt{N*m} = 0.4 \cdot 24 \texttt{N} \cdot \frac{\texttt{8m} + \texttt{6m}}{2}$$

# 12) Total Vertical Load Transmitted to Conical Pivot Bearing for Uniform Pressure

fx 
$$W_{t} = \pi \cdot \left(\frac{D_{s}}{2}\right)^{2} \cdot p_{i}$$

$$1.963495\mathrm{N} = \pi \cdot \left(\frac{0.5\mathrm{m}}{2}\right)^{2} \cdot 10\mathrm{Pa}$$

#### Screw and Nut C

# 13) Force at Circumference of Screw given Helix Angle and Coefficient of Friction

$$\mathbf{fx} \left[ \mathbf{F} = \mathbf{W} \cdot \left( \frac{\sin(\psi) + \mu_{\mathrm{f}} \cdot \cos(\psi)}{\cos(\psi) - \mu_{\mathrm{f}} \cdot \sin(\psi)} \right) 
ight]$$

$$\mathbf{63.89666N} = 60 \text{kg} \cdot \left( \frac{\sin(25^\circ) + 0.4 \cdot \cos(25^\circ)}{\cos(25^\circ) - 0.4 \cdot \sin(25^\circ)} \right)$$

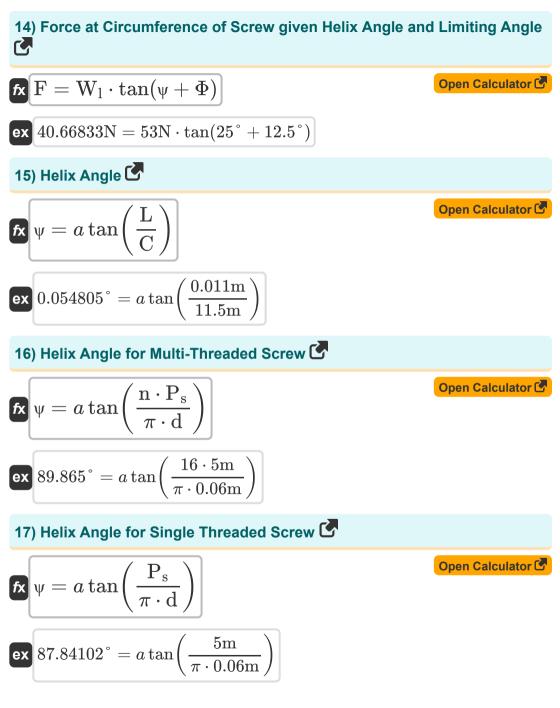




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18) Lead of Screw 🗹

fx 
$$\mathbf{L} = \mathbf{P}_{\mathrm{s}} \cdot \mathbf{n}$$

ex  $80m = 5m \cdot 16$ 

19) Torque Required to Overcome Friction between Screw and Nut

fx 
$$\mathbf{T} = \mathrm{W}_1 \cdot \mathrm{tan}(\psi + \Phi) \cdot rac{\mathrm{d}}{2}$$

ex 
$$1.22005$$
N\*m =  $53$ N · tan $(25^{\circ} + 12.5^{\circ}) \cdot \frac{0.06$ m}{2}

20) Torque Required to Overcome Friction between Screw and Nut while Lowering Load

fx 
$$\mathbf{T} = \mathrm{W}_1 \cdot \mathrm{tan}(\Phi - \psi) \cdot rac{\mathrm{d}}{2}$$

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$$\text{ex} -0.352495 \text{N*m} = 53 \text{N} \cdot an(12.5\degree - 25\degree) \cdot rac{0.06 \text{m}}{2}$$

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#### Screw Jack 🕑

### 21) Efficiency of Screw Jack when Screw Friction as well as Collar Friction Considered

$$f_{\mathbf{X}} \eta = \frac{W \cdot \tan(\psi) \cdot d}{W_1 \cdot \tan(\psi + \Phi) \cdot d + \mu_c \cdot W_1 \cdot R_c}$$

$$e_{\mathbf{X}} 0.643257 = \frac{60 \text{kg} \cdot \tan(25^\circ) \cdot 0.06\text{m}}{53\text{N} \cdot \tan(25^\circ + 12.5^\circ) \cdot 0.06\text{m} + 0.16 \cdot 53\text{N} \cdot 0.02\text{m}}$$

22) Efficiency of Screw Jack when only Screw Friction is Considered 🕑

fx 
$$\eta = rac{ an(\psi)}{ an(\psi+\Phi)}$$

ex 
$$0.607704 = rac{ an(25^\circ)}{ an(25^\circ + 12.5^\circ)}$$

23) Force Required to Lower Load by Screw Jack given Weight of Load 🕑

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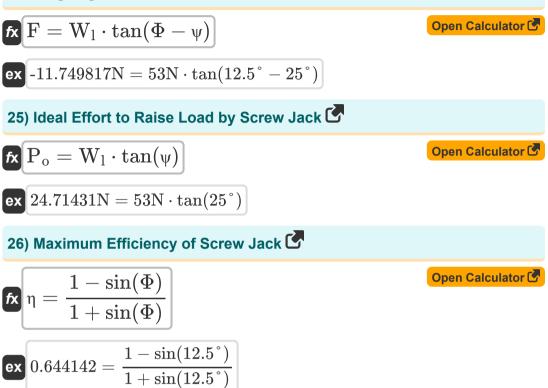
fx 
$$\mathbf{F} = \mathbf{W}_1 \cdot \frac{\mu_{\mathrm{f}} \cdot \cos(\psi) - \sin(\psi)}{\cos(\psi) + \mu_{\mathrm{f}} \cdot \sin(\psi)}$$

$$\begin{array}{l} \textbf{ex} \ -2.961852 \mathrm{N} = 53 \mathrm{N} \cdot \frac{0.4 \cdot \cos(25°) - \sin(25°)}{\cos(25°) + 0.4 \cdot \sin(25°)} \end{array}$$





### 24) Force Required to Lower Load by Screw Jack given weight of load and Limiting angle





#### Variables Used

- C Circumference of Screw (Meter)
- d Mean Diameter of Screw (Meter)
- D<sub>s</sub> Shaft Diameter (Meter)
- F Force Required (Newton)
- h<sub>s</sub> Slant Height (Meter)
- L Lead of Screw (Meter)
- **n** Number of Threads
- **p**<sub>i</sub> Pressure Intensity (Pascal)
- Po Ideal Effort (Newton)
- **P**<sub>s</sub> Pitch (Meter)
- **R** Radius of Bearing Surface (Meter)
- **r**<sub>1</sub> Outer Radius of Bearing Surface (Meter)
- R<sub>1</sub> Outer Radius of Collar (Meter)
- **r**<sub>2</sub> Inner Radius of Bearing Surface (Meter)
- R<sub>2</sub> Inner Radius of Collar (Meter)
- R<sub>c</sub> Mean Radius of Collar (Meter)
- **T** Total Torque (Newton Meter)
- W Weight (Kilogram)
- W<sub>I</sub> Load (Newton)
- W<sub>t</sub> Load Transmitted Over Bearing Surface (Newton)
- α Semi Angle of Cone (Degree)
- η Efficiency

- µ<sub>c</sub> Coefficient of Friction For Collar
- $\mu_f$  Coefficient of Friction
- **Φ** Limiting Angle of Friction (*Degree*)
- **W** Helix Angle (Degree)

#### **Constants, Functions, Measurements used**

- Constant: pi, 3.14159265358979323846264338327950288 Archimedes' constant
- Function: atan, atan(Number) Inverse tan is used to calculate the angle by applying the tangent ratio of the angle, which is the opposite side divided by the adjacent side of the right triangle.
- Function: **cos**, cos(Angle) Cosine of an angle is the ratio of the side adjacent to the angle to the hypotenuse of the triangle.
- Function: cosec, cosec(Angle) The cosecant function is a trigonometric function that is the reciprocal of the sine function.
- Function: **sec**, sec(Angle) Secant is a trigonometric function that is defined ratio of the hypotenuse to the shorter side adjacent to an acute angle (in a right-angled triangle); the reciprocal of a cosine.
- Function: **sin**, sin(Angle) Sine is a trigonometric function that describes the ratio of the length of the opposite side of a right triangle to the length of the hypotenuse.
- Function: tan, tan(Angle) The tangent of an angle is a trigonometric ratio of the length of the side opposite an angle to the length of the side adjacent to an angle in a right triangle.
- Measurement: Length in Meter (m) Length Unit Conversion
- Measurement: Weight in Kilogram (kg) Weight Unit Conversion





- Measurement: Pressure in Pascal (Pa) Pressure Unit Conversion
- Measurement: Force in Newton (N) Force Unit Conversion
- Measurement: Angle in Degree (°) Angle Unit Conversion
- Measurement: Torque in Newton Meter (N\*m) Torque Unit Conversion

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